

# **Towards Textbook Multigrid Efficiency for Fluid Flows : Stagnation**

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# Outline

- Motivation and objective
- Elements of Textbook Multigrid Efficiency (TME)
- Stagnation flow model problems
- Isolation and analysis of difficulties
  - Interior relaxation
  - Boundary closure/relaxation
  - Interactions
- Computational results
- Concluding remarks

## Motivation

- Textbook Multigrid Efficiency (TME) : Solutions attained in a few ( $< 10$ ) minimal work units

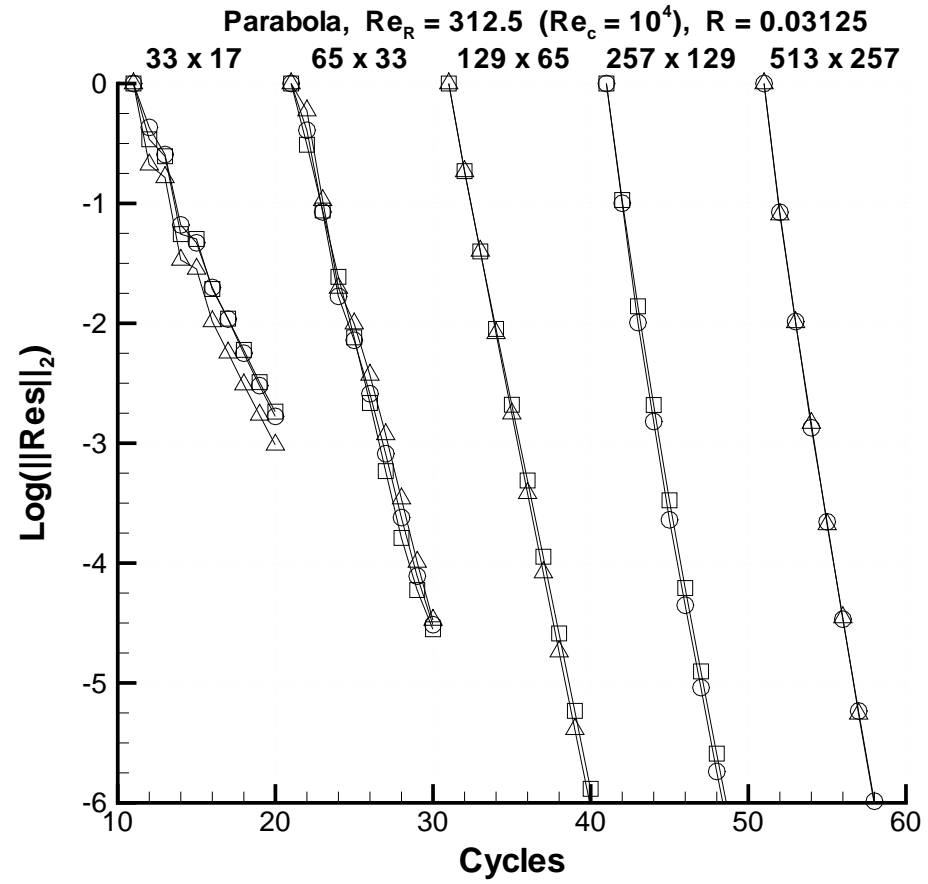
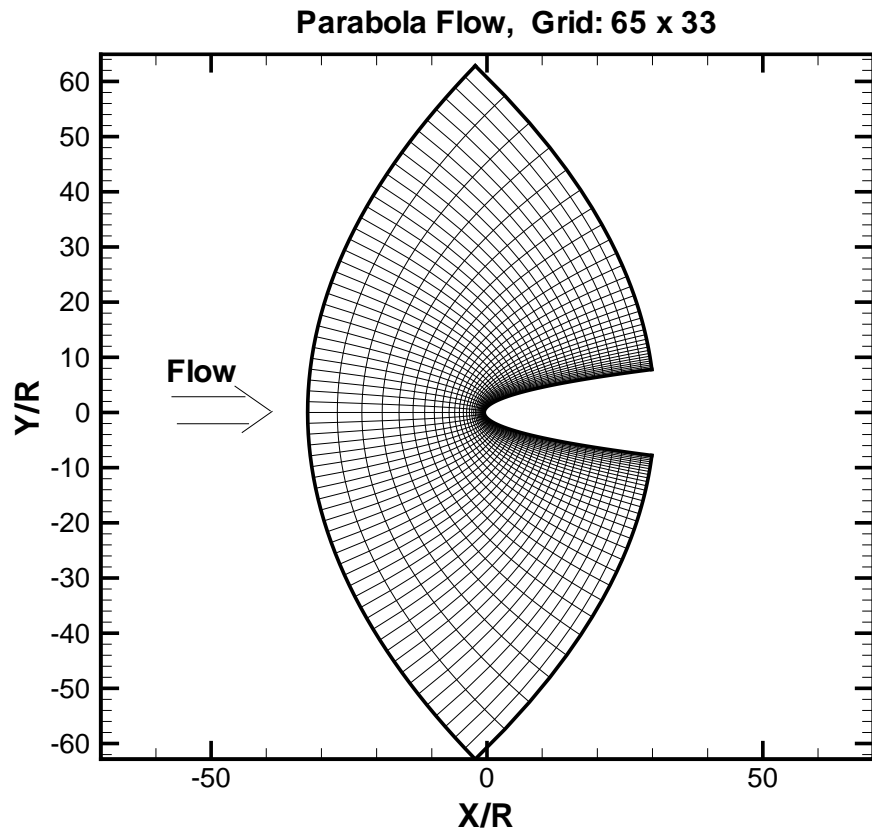
A **minimal work unit** is the operation count in one target-grid residual evaluation

- Motivation : Develop TME algorithms for time-dependent viscous flows applied to flow control applications
  - Builds upon earlier incompressible (Pressure-Poisson) formulations in generalized coordinates
  - Inviscid (Roberts, Sidilkover & Swanson, 1999)
  - Viscous (Swanson, 2001)

## Current Objective

- Current Objective : Overcome difficulties encountered in applications associated with stagnation regions
  - Non-optimal asymptotic convergence rates
  - Poor coarse-grid corrections
  - Reduced efficiency for large domain sizes (coarse grids, high stretching)
- Convergence difficulties also experienced in stagnation for preconditioned compressible formulations (e.g., Turkel, Vatsa & Radespiel, 1996)

# Viscous Flow Around a Parabola



## Incompressible Navier-Stokes (INS) equations

- Pressure-Poisson Formulation:

$$u\partial_x^u u + v\partial_y^u u - \nu\Delta^h u + \partial_x^c p = 0,$$

$$u\partial_x^u v + v\partial_y^u v - \nu\Delta^h v + \partial_y^c p = 0,$$

$$(\partial_x^c u)^2 + 2(\partial_y^c u)(\partial_x^c v) + (\partial_y^c v)^2 + \Delta^h p = 0.$$

- Efficient relaxation over a *major* part of the domain

$$\mathbf{L} \delta \mathbf{q} = -\mathbf{R}^h(\mathbf{q}^n) \quad \delta \mathbf{q} = \mathbf{q}^{n+1} - \mathbf{q}^n$$

- Principal linearization  $\mathbf{L}$  is upper triangular (decoupled relaxation)

## Principal Linearization (From the Standpoint of Relaxation)

- The principal linearization  $\mathbf{L}$  is derived from the full Newton linearization by removing subprincipal terms
- **Scalar equation:** Retain terms that make major contributions to the residual
- **Systems:** Retain terms that make major contributions to the *determinant* of the matrix operator

## Elements of TME : Principal Linearization

(Example - Nonlinear Convection Operator:  $u\partial_x^h u$ )

Full Linearization:  $[u\partial_x^h + \partial_x^h u]\delta u$

High-Frequency Contribution (local mode analysis):

$$\frac{1}{h} \left[ \widehat{u\partial_x^h} + h\partial_x^h u \right] \delta u$$

$$O(u) \quad O(hu_x)$$

Regular flow region (  $hu_x < u$  ) :  $L = u\partial_x^h$

Stagnation flow region (  $hu_x \approx u$  ) :  $L = u\partial_x^h + (\partial_x^h u)$



## Principal Linearization

- Principal Linearization in the **regular flow field**:

$$\mathbf{L} = \begin{bmatrix} Q^u & 0 & 0 \\ 0 & Q^u & 0 \\ 0 & 0 & \Delta^h \end{bmatrix}$$

$$Q^u = u\partial_x^u + v\partial_y^u - \nu\Delta^h$$

$$\det \mathbf{L} = (Q^u)^2 \Delta^h$$

- Principal Linearization Near Stagnation (Full Linearization):

$$\mathbf{L} = \begin{bmatrix} Q^u + (\partial_x^u u) & (\partial_y^u u) & \partial_x^c \\ (\partial_x^u v) & Q^u + (\partial_y^u v) & \partial_y^c \\ 2(\partial_x^c u)\partial_x^c + 2(\partial_x^c v)\partial_y^c & 2(\partial_y^c u)\partial_x^c + 2(\partial_y^c v)\partial_y^c & \Delta^h \end{bmatrix}$$

## Relaxation Strategy for Factorizable Scheme

- The efficiency goal is the “per relaxation” convergence rates associated with scalar factors in the regular flow field
- Global : decoupled relaxation in the regular flow field
- Local : coupled relaxation near boundaries/singularities
  - The **local** principal linearization is closer to a full linearization and generally not easily analyzed
  - A different discretization might be used
- General approach is to tailor **local** relaxation to **local** properties of the solution

## Relaxation Framework

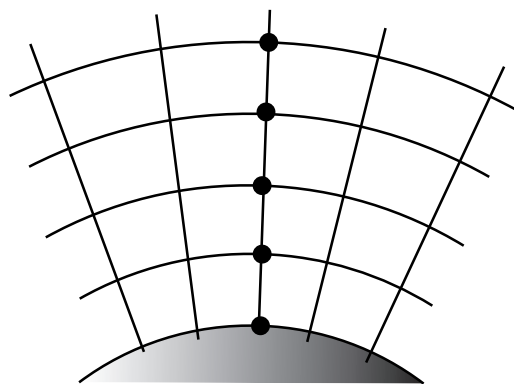
- Linearized equations relaxed at each level

$$\mathbf{L}\delta(\delta\mathbf{q}) = -\mathbf{R}(q) - \frac{\partial\mathbf{R}}{\partial\mathbf{q}}\delta\mathbf{q}$$

- Global Relaxation : alternating line-implicit
  - Pressure-equation Gauss-Seidel relaxation (velocity is fixed)
  - Relaxation (marching) of momentum equations
- Local Relaxation: equations are solved simultaneously
  - at 5 points near the body surface or outflow
  - at 3 points near the inflow boundary

# Coupling of Global and Local Relaxation (At a Boundary)

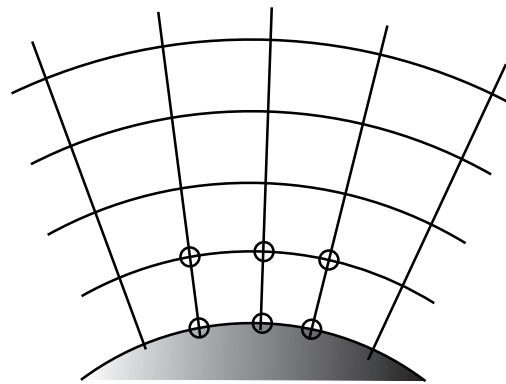
Global Relaxation  
Line Gauss-Seidel for  $p$



●  $p$

+

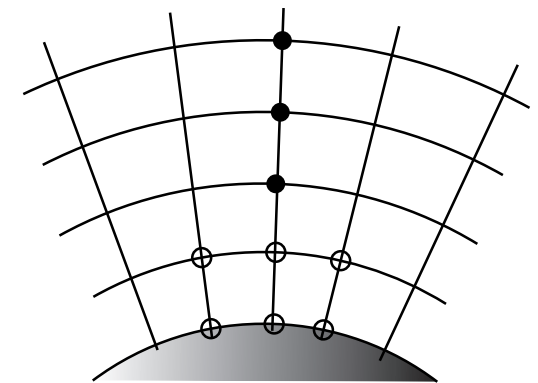
Local Relaxation  
Block Collective for  $u, v,$  and  $p$



○  $u, v, p$

=

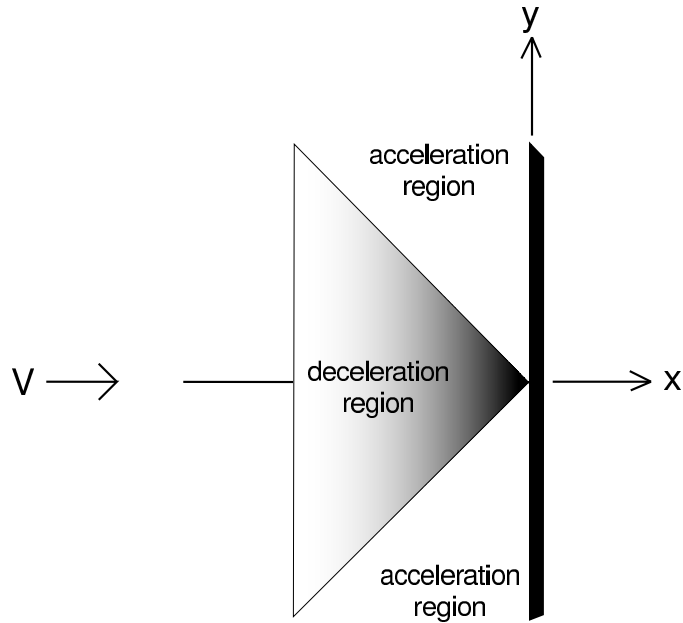
Coupled Relaxation



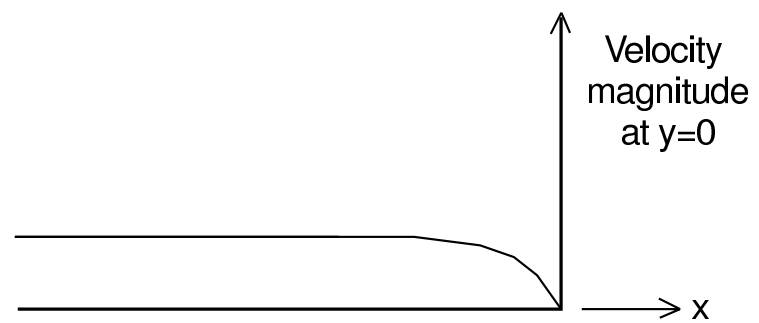
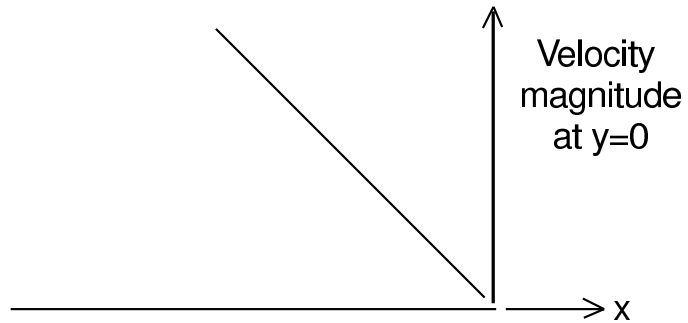
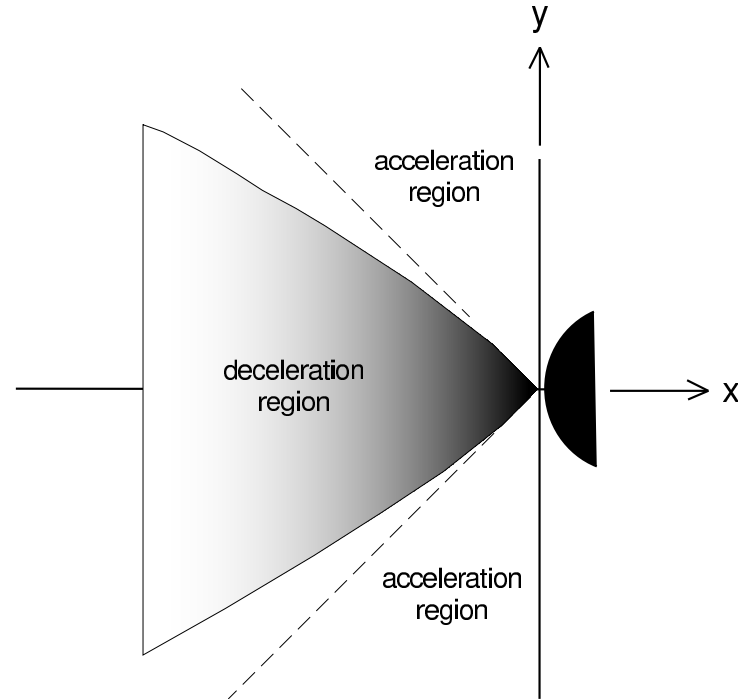
●  $p$   
○  $u, v, p$

# Stagnation Flow Model Problems

Plane Stagnation



Circular Cylinder Stagnation

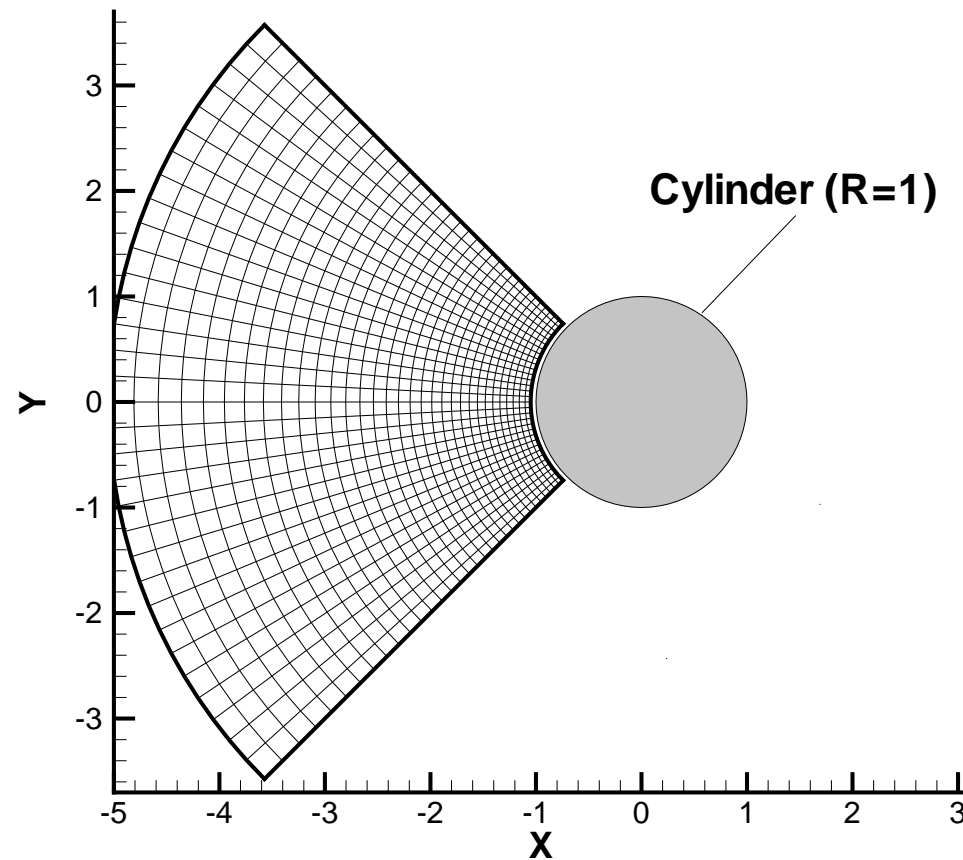


## Inviscid Stagnation Flow : Boundary Conditions

- Differential Conditions I - primitive equation set:
  - Inflow : Given  $u, v$
  - Outflow : Given  $p$
  - Tangency : Zero normal velocity
- Differential Conditions II - Pressure-Poisson formulation:
  - Inflow : Continuity equation,  $u_x + v_y = 0$
  - Continuity equation enforced only to within discretization error:  $Q(u_x + v_y) = 0$
- Other necessary considerations
  - Numerical closure equations
  - Conservation law equations at tangency boundaries

# An Example of the Difficulties Near Stagnation

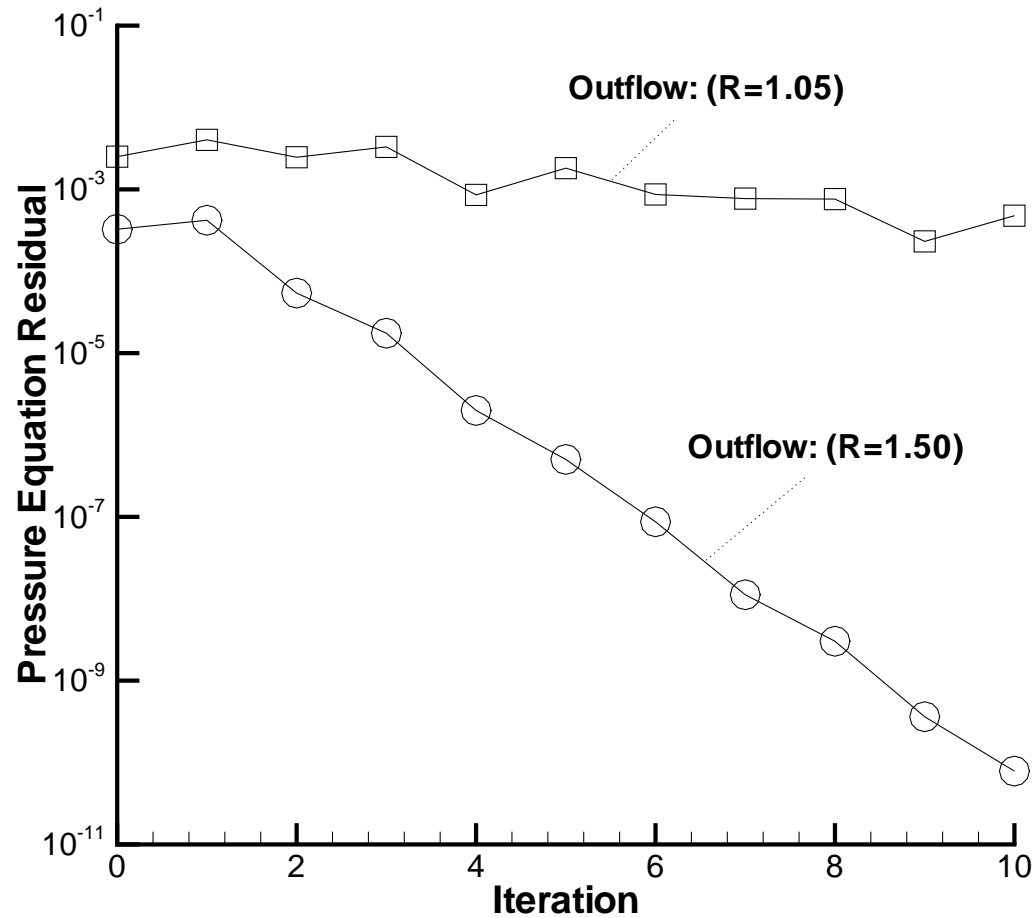
Outflow Boundary Approaching Cylinder  
Upper Triangular Solves + Boundary Solves



Shown above : Inflow @  $R = 5$  ; Outflow @  $R = 1.05$

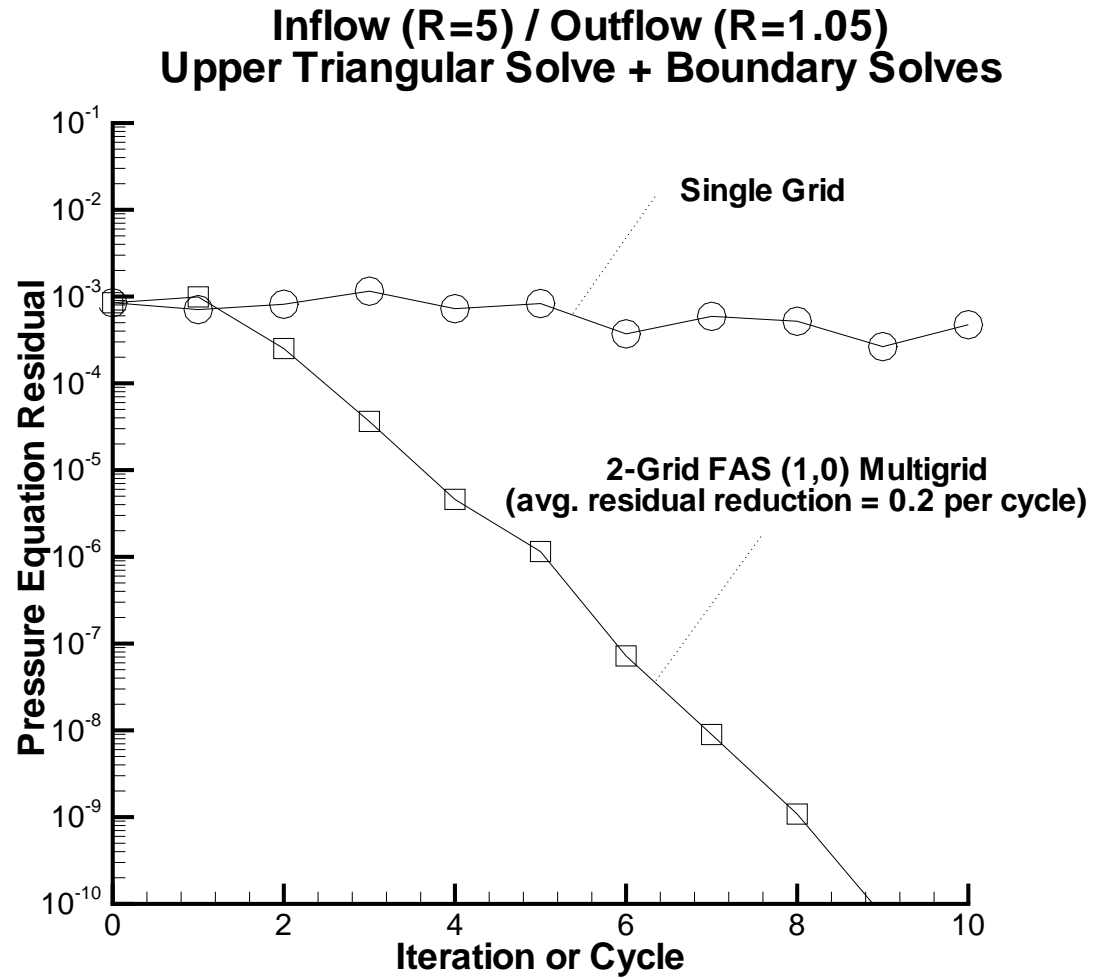
# An Example of the Difficulties Near Stagnation

Outflow Boundary Approaching Cylinder  
 Upper Triangular Solves + Boundary Solves





# An Example of the Difficulties Near Stagnation



## Analysis of Relaxation : Plane Stagnation

Exact Solution:  $u = -x > 0$ ,  $v = +y > 0$ ,  $u_x = -1$ ,  $v_y = +1$

- Examine behavior of upper triangular **solve**  
(**L** in **regular flow** + readily available terms)

$$\mathbf{L} = \begin{bmatrix} Q^u + u_x & 0 & \partial_x^c \\ 0 & Q^u + v_y & \partial_y^c \\ 0 & 0 & \Delta^h \end{bmatrix}$$

- Analysis of iteration is a variable coefficient problem

$$\mathbf{L} \delta \epsilon = -\frac{\partial \mathbf{R}}{\partial \mathbf{q}}(\epsilon^n) \quad \delta \epsilon = \epsilon^{n+1} - \epsilon^n$$

- Several constant coefficient approximations analyzed

## Forms of Analysis : Plane Stagnation

- Full space (Fourier) analysis : prediction of possible worst amplifications at initial iterations
- Mode analysis with boundary conditions : prediction of asymptotic convergence rates and penetration distances for characteristic (velocity) direction
  - Differential equations only
  - Periodic in single direction
- Domain size  $L$  constrained for relevancy of *constant* coefficient approximation to *variable* coefficient problem

$$|u_x L| \leq u \quad |v_y L| \leq v$$

## Results of Full Space Analysis

- Eigenvalues of error amplification matrix are  $f(N, v/u)$ :

$$\lambda = \frac{2}{\theta_x^2 + \theta_y^2} \left[ \frac{v_y \theta_y^2}{\widehat{Q} + v_y} + \frac{u_x \theta_x^2}{\widehat{Q} + u_x} \right] \quad \theta_x, \theta_y \in [-\pi, \pi]$$

- Uniformly good smoothing rates were found (maximum 1/10)
- In cross-characteristic direction (normal to velocity), the lowest frequency contribution becomes:

$$\lim_{N \rightarrow \infty} \widehat{Q} = 0 \quad \Rightarrow \quad \lambda = 2$$

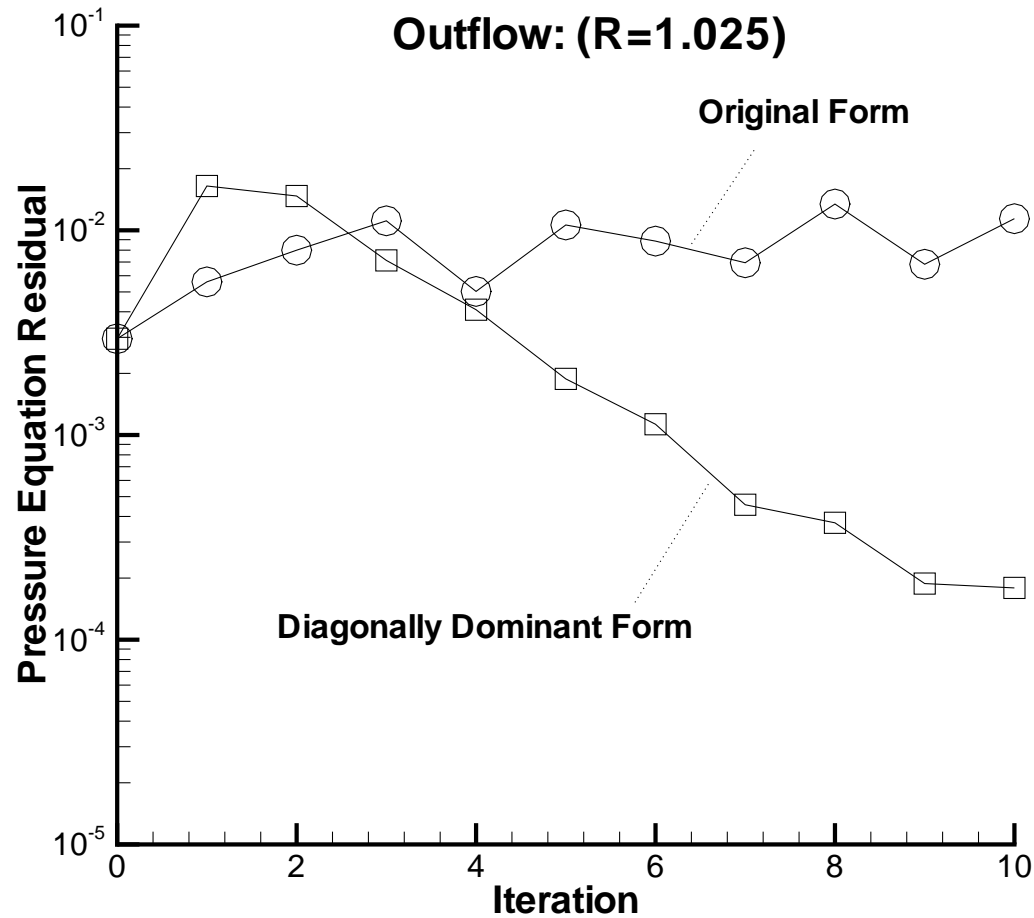
## Results of Mode Analysis with Boundary Conditions

- Fast asymptotic error decay for  $v = 0$
- Initial error amplified for a few steps
- Analysis prompted assessment of diagonally dominant  $\mathbf{L}$  for momentum equations
  - Along symmetry plane,  $|\mathbf{L}| = \left| \frac{\partial \mathbf{R}}{\partial \mathbf{q}} \right| = (Q - u_x) \Delta$
  - $(\mathbf{L})_{1,1} = Q + |u_x|$        $(\mathbf{L})_{2,2} = Q + |v_y|$
  - Implementation was somewhat effective in eliminating instability of nonlinear problem (both plane and cylinder stagnation)

# An Example of the Difficulties Near Stagnation

Outflow Boundary Approaching Cylinder

Upper Triangular Solves



# Numerical Closure Equations : Outflow Boundary

(Pressure Specified at  $x = \text{Constant}$ )

- Original closure :

- Upwind discretization of momentum equations

$$Q^u u + \partial_x^u p = 0 \quad Q^v v + \partial_y^c p = 0$$

- Revised closure :

- Upwind discretization of v-momentum equation

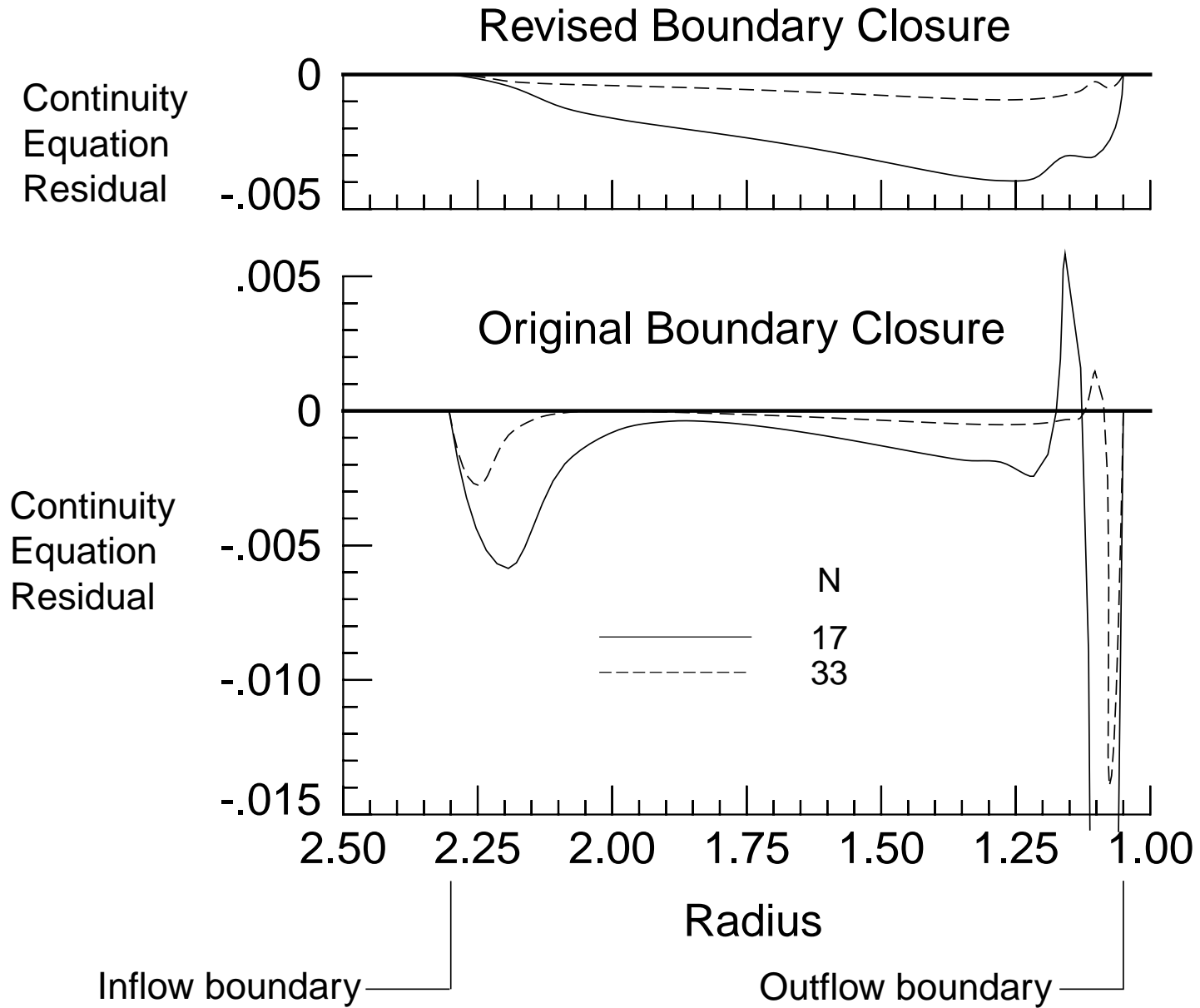
$$Q^u v + \partial_y^c p = 0$$

- Compact discretization of continuity equation

$$\partial_x^c u + \partial_y^c v = 0$$

- Both conditions yield asymptotically accurate schemes but revised closure allows computation on much coarser grids

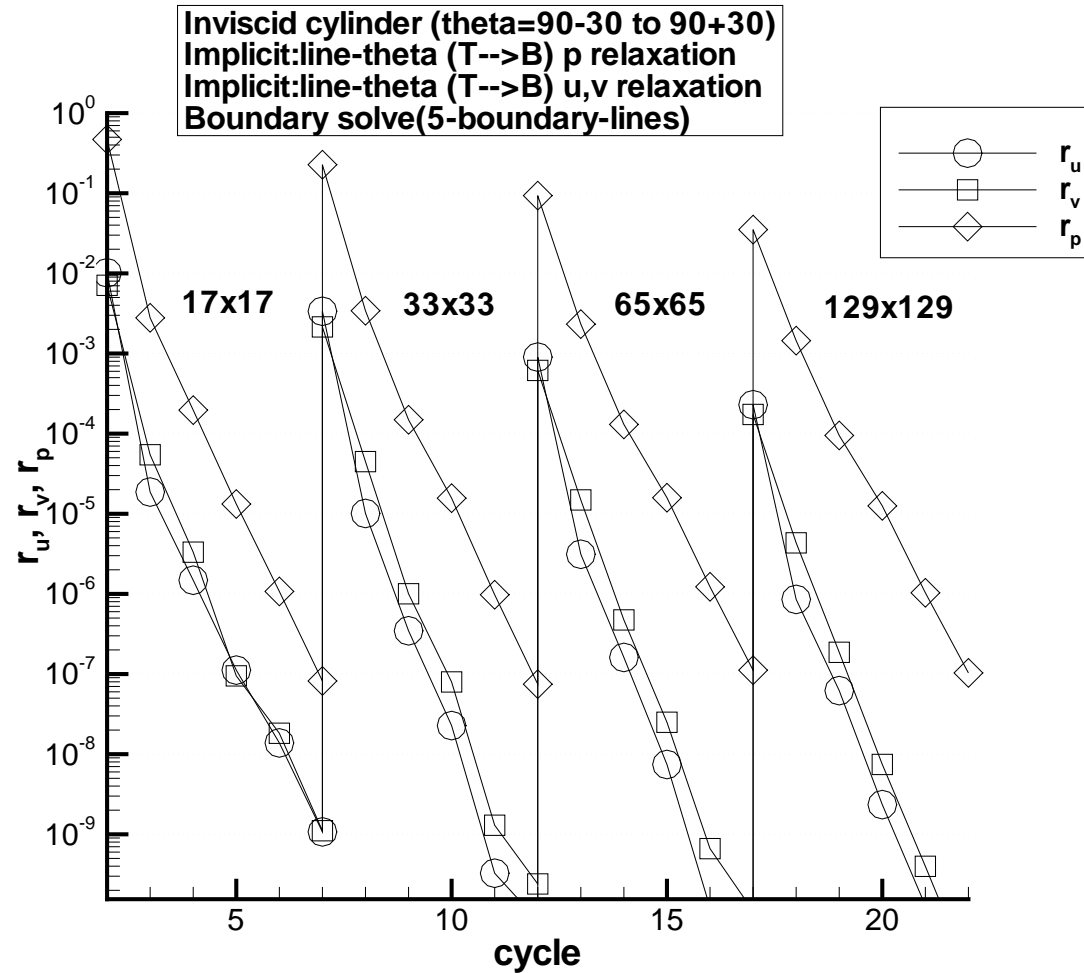
# Effect of Discrete Boundary Closure





# Stagnation Cylinder Flow Residual Convergence

$FV(2, 1)$  cycle



# Stagnation Flow

## FMG-1 Algorithm, Algebraic Error

Grid	Discretization Error	Alg./Discr. Err
17x17	.320E-03	0.24
33x33	.756E-04	0.36
65x65	.182E-04	0.55
129x129	.456E-05	0.6

## Concluding Remarks

- Several issues isolated and improved in stagnation flows
  - Discrete closure conditions at boundary
  - Instability of decoupled solves in deceleration region
- Improved relaxations developed
  - Interior uncoupled line relaxations locally coupled more strongly at boundary
  - Diagonally dominant relaxation for momentum equations
- TME efficiency attained (10 minimum work units)
  - FMG-1 algebraic errors below discretization errors
  - Asymptotic rate of elliptic factor attained